

# Conjugate free convection in a vertical channel filled with nanofluid

Flavius Pătrulescu and Teodor Groşan

**Abstract.** The steady natural conjugate convection in a long vertical channel filled with a nanofluid and including internal heat generation is presented in this paper. A new mathematical model is proposed for the momentum, energy and nanoparticles' concentration equations. The system of partial differential equations is written in terms of dimensionless velocity, temperature and concentration of the nanoparticles and is solved analytically. The effects of the governing parameters, such as the ratio between the thermophoresis parameter and the Brownian motion parameter,  $R$ , and the buoyancy ratio parameter,  $Nr$ , on the velocity, temperature and nanoparticles' concentration are studied. It is found that the addition of the nanoparticles into the fluid reduces the temperature and enhances the heat transfer. A limit case when the thermal conductivity of the nanoparticles is much larger than the thermal conductivity of the base fluid has been also studied.

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## 1. Introduction

General fluids used in industrial processes involving heat transfer (energy generation, insulation, cooling of microelectronic components) are water, mineral oil, ethylene glycol, etc. (see [3], [11]). Low values of the physical properties of these fluids (thermal conductivity, density, expansion coefficient, etc.) limit the efficiency of heat transport and it is necessary to obtain new type of fluids, having improved heat transfer characteristics (see [8]). In order to enhance the thermal characteristics of the fluids, one can form mixtures by adding ultra-fine solid particles (metallic, non-metallic or polymeric) to the fluid. Choi [7] was probably, the first one who called the fluids with particles of nanometer dimensions nanofluids.

Over the last twenty years, many industrial processes, biology, medicine, catalytic chemistry and environmental applications started to use nanotechnologies (see

[13] and [17]). Different mathematical models were used by several authors to describe heat transfer in nanofluids. Among these models the most used are those where the concentration of nanoparticles is constant and the addition of nanoparticles into the base fluid improved their physical properties (see [18]). Moreover, other models based on physical properties variation include thermal dispersion (see [12]) or Brownian motion (see [14]). A more complex mathematical model (see [4]) considers that nanoparticles' concentration is variable and incorporates the effects of Brownian motion and thermophoresis. Recently, Celli [6] had the idea to combine the model proposed by Buongiorno [4] and the model based on improved physical properties considering for the last one an average concentration of nanoparticles.

Many times theoretical problems as well as industrial processes and natural phenomena are modelled using simple geometries such as infinite channels (see [1], [2]). However, in real simulation it is necessary to take into account the interaction between the convective heat transfer in nanofluid and conductive heat transfer in the thick solid walls. Such situations (i.e. conjugate heat transfer) appears in cellular structures, cavities or channels with solid walls, etc.

Several authors such as Pătrulescu and Groșan [16], Groșan [9], Groșan and Pop [10] and Li [15] have studied the fully developed flow in a vertical channel filled by a nanofluid using different mathematical models for nanofluid and different boundary conditions. In the present paper, the fully developed conjugate heat transfer in a vertical channel filled with a nanofluid when heat generation in the solid wall is considered has been studied analytically.

## 2. Notations and preliminaries

Consider the fully developed steady conjugate free convection flow of an incompressible nanofluid in vertical channel differentially heated. The left wall of the channel is kept at a constant temperature  $T_H$ , while the right wall has a constant temperature  $T_C$ . We consider a two-dimensional coordinate frame in which  $x$ -axis is aligned vertically upwards, see Figure 1. The left wall is at  $y = 0$  and has thickness  $b$ ,  $b > 0$ . The right wall is at  $y = L$ ,  $L > b$ .

The field variables are the temperature in solid domain  $T_s$ , the temperature in nanofluid domain  $T_f$ , the velocity  $\mathbf{v} = (u, v)$  and the nanoparticle volume fraction  $C$ .

As in [15] it is assumed that the nanoparticles' flux

$$q_c = D_B \nabla C + (D_T/T_f) \nabla T_f \quad (2.1)$$

is zero on the solid vertical walls. In (2.1)  $D_B$  and  $D_T$  are the Brownian and thermophoretic diffusion coefficients.

To define the effective viscosity we use a model proposed in [5], namely

$$\mu_{nf} = \frac{\mu_f}{(1 - C_0)^{2.5}}, \quad (2.2)$$

where  $\mu_f$  represents the dynamic viscosity of base fluid and  $C_0$  is the reference nanoparticles volume fraction concentration. Moreover, the effective thermal conductivity is approximated by a model introduced in [19], namely

$$k_{nf} = k_f \frac{k_p + 2k_f - 2C_0(k_f - k_p)}{k_p + 2k_f + C_0(k_f - k_p)}, \tag{2.3}$$

where  $k_f$  and  $k_p$  are the thermal conductivity of the base fluid and thermal conductivity of the nanoparticles, respectively. The behavior of  $\mu_{nf}$  and  $k_{nf}$  as functions of  $C_0$  was studied in [6] for a side heated square cavity. There, the nanofluid is composed of water as base fluid and Alumina as nanoparticles dispersed inside the base fluid.

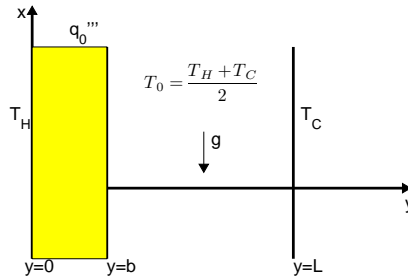


FIGURE 1. Geometry of the problem and the co-ordinate system

The limit case of  $k_{nf}$  is

$$k_{nf}^{lim} = k_f \frac{1 + 2C_0}{1 - C_0} \tag{2.4}$$

and it is obtained when the ratio  $k_f/k_p$  is very small. Physically this is possible when high thermal conductivity nanoparticles such as copper, gold, carbon nanotubes are used. Thus, it is possible to predict the maximum achievable temperature in this mathematical model for different kind of fluids and concentrations of nanoparticles. Some examples concerning nanofluids' thermo-physical properties are shown in the following table.

Physical properties	Fluid phase (water)	Cu	$Al_2O_3$	$TiO_3$
$C_p$ (J/kgK)	4179	385	765	686.2
$\rho$ ( $kg/m^3$ )	997.1	8933	3970	4250
$k$ (W/mK)	0.613	400	40	8.9538
$\beta \times 10^{-5}$ (1/K)	21	1.67	0.85	0.9

### 3. Basic equations

In this section we provide the governing equations for the flow and heat transfer. Thus, as in [16], we consider the following equation for temperature in solid domain

$$\alpha_s \nabla^2 T_s + \frac{q_0'''}{(\rho c)_s} = 0, \quad (3.1)$$

where  $q_0'''$  represents the heat generation,  $\alpha_s$  is the thermal diffusivity coefficient and  $(\rho c)_s$  represents the heat capacity. As in [10], in the fluid domain we consider the following four field equations in the vectorial form. More exactly, the equations embody the conservation of total mass, momentum, thermal energy and nanoparticles' concentration. Thus, we have

$$\nabla \cdot \mathbf{v} = 0, \quad (3.2)$$

$$\rho_f (\mathbf{v} \cdot \nabla \mathbf{v}) = \mu_{nf} \nabla^2 \mathbf{v} + \{\rho_p C + (1 - C)[\rho_f (1 - \beta(T_f - T_0))]\} \mathbf{g}, \quad (3.3)$$

$$(\rho c)_f (\mathbf{v} \cdot \nabla T_f) = k_{nf} \nabla^2 T_f + (\rho c)_p [D_B \nabla T_f \cdot \nabla C + (D_T/T_f) \nabla T_f \cdot \nabla T_f], \quad (3.4)$$

$$\mathbf{v} \cdot \nabla C = \nabla (D_B \nabla C + (D_T/T_f) \nabla T_f). \quad (3.5)$$

Here  $\rho_f$  is the fluid density,  $\rho_p$  is the nanoparticle mass density,  $\beta$  represents the thermal expansion coefficient,  $\mathbf{g}$  is the gravitational acceleration. Finally,  $(\rho c)_f$  and  $(\rho c)_p$  are the heat capacity of the base fluid and of the nanoparticle material, respectively.

In the rest of the paper we use the following linearized version of the momentum equation (see [10])

$$\rho_f (\mathbf{v} \cdot \nabla \mathbf{v}) = \mu_{nf} \nabla^2 \mathbf{v} + [(\rho_p - \rho_{f_0})(C - C_0) - (1 - C_0)\rho_{f_0}\beta(T_f - T_0)] \mathbf{g}, \quad (3.6)$$

where  $\rho_{f_0}$  represents the reference fluid density.

Based on the fact that the flow is fully developed we introduce the following assumptions

$$v = 0, \quad \frac{\partial T_f}{\partial x} = 0, \quad \frac{\partial T_s}{\partial x} = 0, \quad \frac{\partial C}{\partial x} = 0. \quad (3.7)$$

Taking into account (2.1) and (3.7) the governing equations for the flow and heat transfer (3.1)–(3.6) become

$$\alpha_s \frac{d^2 T_s}{dy^2} + \frac{q_0'''}{(\rho c)_s} = 0, \quad (3.8)$$

$$\frac{d^2 T_f}{dy^2} = 0, \quad (3.9)$$

$$D_B \frac{dC}{dy} + \frac{D_T}{T_f} \frac{dT_f}{dy} = 0, \quad (3.10)$$

$$\mu_{nf} \frac{d^2 u}{dy^2} + (1 - C_0)\rho_{f_0}\beta(T_f - T_0)g - (\rho_p - \rho_{f_0})(C - C_0)g = 0, \quad (3.11)$$

subject to the boundary conditions

$$T_s|_{y=0} = T_H, \quad T_f|_{y=L} = T_C, \quad (3.12)$$

$$T_f|_{y=b} = T_s|_{y=b}, \quad (3.13)$$

$$k_s \frac{dT_s}{dy}|_{y=b} = k_{nf} \frac{dT_f}{dy}|_{y=b}, \quad (3.14)$$

$$u(b) = u(L) = 0. \tag{3.15}$$

In (3.14)  $k_s$  denotes the thermal conductivity of solid domain. To complete the set of equations and boundary conditions we add the following nanoparticles' conservation relation

$$\int_b^L C(y) dy = Q_0, \tag{3.16}$$

where  $Q_0$  is defined in the next section.

#### 4. Dimensionless equations

In this section we solve equations (3.8)–(3.11) subject to (3.12)–(3.16). To this end, we consider the following dimensionless variables used in [16].

$$Y = \frac{y}{L}, \Theta_s = \frac{k_s(T_s - T_0)}{q_0'''L^2}, \Theta_f = \frac{k_s(T_f - T_0)}{q_0'''L^2}, \phi = \frac{C - C_0}{C_0}, U = \frac{u}{U_c}, \tag{4.1}$$

where  $U_c$  is the characteristic velocity given by

$$U_c = \frac{g\beta(\frac{q_0'''L^2}{k_s})L^2}{\nu_f}. \tag{4.2}$$

In (4.2)  $\nu_f$  represents the kinematic viscosity of base fluid.

We substitute dimensionless variables (4.1) into equations (3.8)–(3.11) and we obtain the following ordinary differential equations

$$\frac{d^2\Theta_s}{dY^2} + 1 = 0, \tag{4.3}$$

$$\frac{d^2\Theta_f}{dY^2} = 0, \tag{4.4}$$

$$\frac{d\phi}{dY} + \frac{R}{w\Theta_f + 1} \frac{d\Theta_f}{dY} = 0, \tag{4.5}$$

$$\frac{d^2U}{dY^2} = Nr(1 - C_0)^{2.5}\phi - (1 - C_0)^{3.5}\Theta_f, \tag{4.6}$$

where  $w$  is a dimensionless constant given by

$$w = \frac{q_0'''L^2}{k_sT_0} = \frac{2q_0'''L^2}{k_s(T_H + T_C)} \tag{4.7}$$

and, as in [10],  $Nr$  is the buoyancy ratio parameter defined by

$$Nr = \frac{g(\rho_p - \rho_{f0})C_0L^2}{\mu_fU_c}. \tag{4.8}$$

Moreover,  $R$  is given by

$$R = \frac{N_t}{N_b}, \tag{4.9}$$

and it represents the ration between the thermophoresis parameter and the Brownian motion parameter (see [10]).

The boundary conditions (3.12)–(3.15) become

$$\Theta_s|_{Y=0} = q, \quad (4.10)$$

$$\Theta_s|_{Y=r} = \Theta_f|_{Y=r}, \quad (4.11)$$

$$\frac{d\Theta_s}{dY}|_{Y=r} = K \frac{d\Theta_f}{dY}|_{Y=r}, \quad (4.12)$$

$$\Theta_f|_{Y=1} = -q, \quad (4.13)$$

$$U(r) = U(1) = 0, \quad (4.14)$$

where  $r$ ,  $q$ ,  $K$  are given by

$$r = \frac{b}{L}, \quad q = \frac{k_s(T_H - T_C)}{2q_0''L^2}, \quad K = \frac{k_{nf}}{k_s}. \quad (4.15)$$

Finally, we choose  $Q_0 = C_0L(1 - r)$  such that (3.16) takes the form

$$\int_r^1 \phi(Y)dY = 0. \quad (4.16)$$

## 5. Results and discussions

In this section we determine the exact solutions of equations (4.3)–(4.6) and discuss the results. Integrating equations (4.3), (4.4) and taking into account boundary conditions (4.10)–(4.13) we obtain

$$\Theta_s(Y) = -\frac{Y^2}{2} + A_1Y + A_2, \quad (5.1)$$

and

$$\Theta_f(Y) = A_3Y + A_4. \quad (5.2)$$

The constants  $A_1$ ,  $A_2$ ,  $A_3$ ,  $A_4$  are given by

$$A_1 = r + K \frac{\frac{1}{2}r^2 + 2q}{r(1 - K) - 1}, \quad A_2 = q, \quad (5.3)$$

and

$$A_3 = \frac{\frac{1}{2}r^2 + 2q}{r(1 - K) - 1}, \quad A_4 = -q - A_3. \quad (5.4)$$

Moreover, integrating (4.5) and taking into account (5.2), (5.4) and (4.16) we obtain

$$\phi(Y) = -\frac{R}{w} \left( \ln(wA_3Y + wA_4 + 1) - \frac{C_a}{1 - r} \right). \quad (5.5)$$

The constant  $C_a$  is given by

$$C_a = (1 + z) \ln(wA_3 + wA_4 + 1) - (r + z) \ln(wA_3r + wA_4 + 1) - (1 - r),$$

where

$$z = \frac{1 + wA_4}{wA_3}.$$

Finally, integrating (4.6) and taking into account (5.2), (5.4), (5.5) and boundary conditions (4.14) we obtain

$$U(Y) = G(Y) + C_1Y + C_2. \quad (5.6)$$

The constants  $C_1, C_2$  are given by

$$C_1 = \frac{G(r) - G(1)}{1 - r}, \quad C_2 = \frac{rG(1) - G(r)}{1 - r}, \tag{5.7}$$

and function  $G$  is defined in the following way

$$G(Y) = (1 - C_0)^{2.5} \left[ -C_b \left( \frac{Y^2}{2} + zY + \frac{z^2}{2} \right) \ln(wA_3Y + wA_4 + 1) \right. \\ \left. - (1 - C_0)A_3 \frac{Y^3}{6} + \left( \frac{3}{2}C_b + \frac{C_aC_b}{1 - r} - (1 - C_0)A_4 \right) \frac{Y^2}{2} + zC_b \frac{Y}{2} \right],$$

where

$$C_b = \frac{RN_r}{w}.$$

Next we determine the Nusselt number. For the conjugate wall is defined as:

$$Nu = \frac{hL}{k_f} \Big|_{y=b}, \tag{5.8}$$

where the convective heat transfer coefficient,  $h$ , is obtained from relation (see [16]):

$$-k_{nf} \frac{dT_f}{dy} \Big|_{y=b} = h(T_f|_{y=b} - T_0). \tag{5.9}$$

Substituting (5.9) in (5.8) the dimensionless form of Nusselt number becomes:

$$Nu = -\frac{k_{nf}}{k_f} \frac{1}{\Theta_f|_{Y=r}} \frac{d\Theta_f}{dY} \Big|_{Y=r}. \tag{5.10}$$

Tacking into account (5.2) we deduce that

$$Nu = -K \frac{A_3}{A_3r + A_4}. \tag{5.11}$$

Finally, we present the effects of the governing parameters on the velocity, temperature and nanoparticles' concentration. In this study we consider the following fixed values  $r = 0.3, T_H = 300, T_C = 15, k_s = 1.2, k_f = 0.613, C_0 = 0.08, q = 1$ . Figure 2 presents the variation of the solid temperature profiles for different kind of nanoparticles. It is obvious that the addition of the nanoparticles leads to a decrease of the temperature in solid, specially on the solid-fluid interface and the results for copper nanoparticles are the most close to the limit case. The influence of the parameter  $R$  on the nanoparticles' concentration profiles in the limit case is depicted in Figure 3. The concentration profile is almost flat for small values of the parameter  $R$ , while for large values large differences between the concentration values on the left and on the right walls appear. The variation of the velocity profiles with  $R$  and  $Nr$  is given in Figures 4 and 5, respectively. In both cases there is a reversed flow near the right wall. The maximum velocity increases with the increase of  $R$  and  $Nr$ , respectively.

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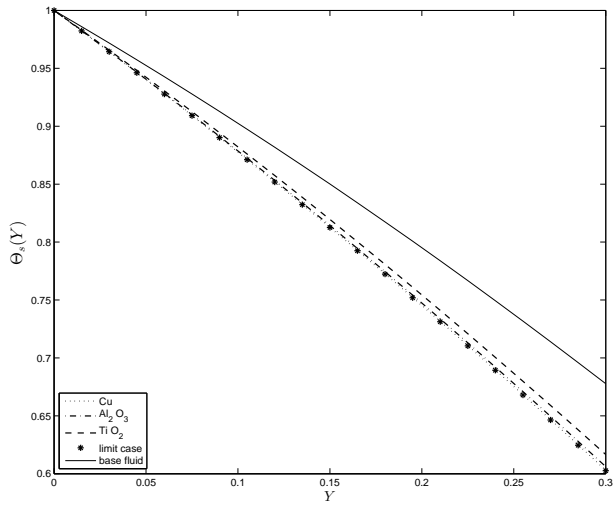


FIGURE 2. Solid temperature

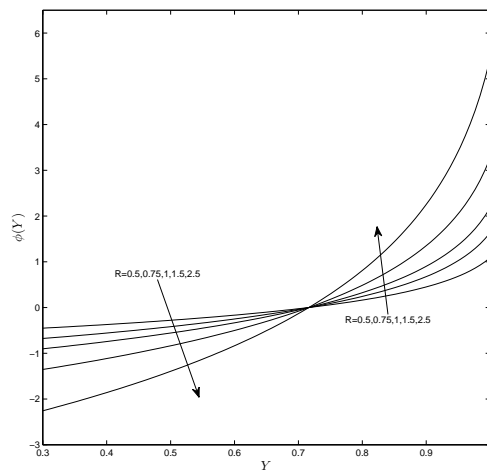


FIGURE 3. Nanoparticles' concentration



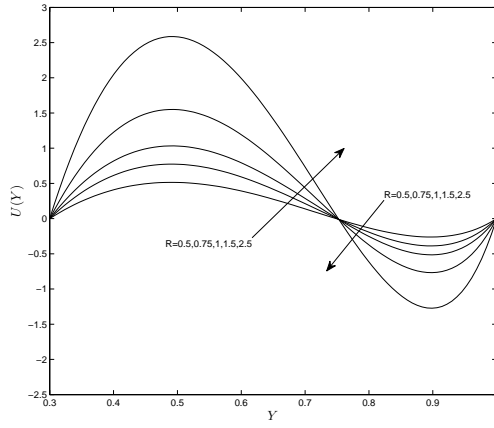


FIGURE 4. Velocity profile,  $N_r = 100$

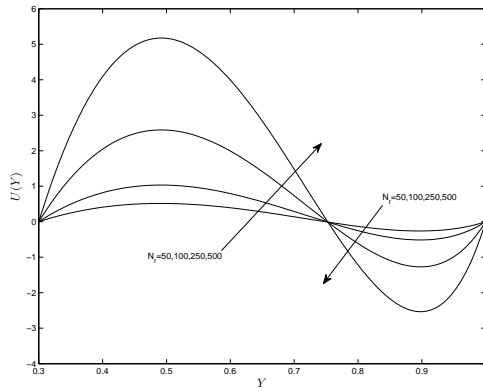


FIGURE 5. Velocity profile,  $R = 1$

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